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Research Article

Experimental investigation of screens as energy dissipaters in submerged hydraulic jump

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Abstract: This study investigated the effectiveness of screens as energy dissipaters in small hydraulic structures using physical experiments. In the experimental design, the Froude number, screen arrangements, and screen porosity were the major controlling parameters. The experiments covered a range of Froude numbers between 2.5 and 8.5, screen porosity of 40% and 50%, and gaps of double screens between 1 and 5 cm. The main goal of this study was investigation of the screens creating submerged hydraulic jumps. The flow depth was digitally measured in order to perceive the turbulences and validate the results of energy dissipation against those of the methods in the literature. The experimental results show the importance of each parameter on the screen performance. The screens with double arrangement with the imposed hydraulic jump dissipated more energy. The gaps of the double arrangement had an insignificant effect on energy dissipation. The double screen with porosity of 40% had the best performance. Since the Froude number was found to be the most effective parameter, quadratic equations as a function of this number were fitted to the experimental results with high \mathbb{R}^2 and very low RMSE values. The quadratic equations can be employed as interpolators and/or extrapolators.

Key words: Energy dissipation, screens, Froude number, submerged hydraulic jump, imposed hydraulic jump

1. Introduction

Control of velocity and consequently energy of water is one of the fundamental problems in hydraulic engineering. Continuous energy transformation takes place with the flow of water in an environment. The understanding of this energy transformation process is crucial to hydraulic engineers.

There have been theoretical and experimental studies to understand the energy transformation process and consequently the applications of energy dissipaters [1–4]. As is well known, one of the energy dissipaters is the creation of hydraulic jump. The hydraulic structural design causing a jump is not only an energy dissipater but also acts as a control structure.

In addition to hydraulic jump-type energy dissipaters, there are impact-type ones as well. The impact-type dissipaters include drops [2,3], baffled outlets (also known as the impact basin - USBR type VI), vertical stilling wells, and sky jumps [4]. Peterka [5] investigated the design of a flip bucket as an impact-type energy dissipater. Mason [6] outlined guidelines for impact-type energy dissipaters. Espinoza and Zevallos

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[7] investigated impact-type dissipaters from a scouring point of view while Schmocker et al. [8] investigated them from an aeration perspective.

This study investigates screens as an energy dissipater at the downstream side of small hydraulic structures. The screen and its usage as an energy dissipater in the downstream of hydraulic structures were first introduced by Rajaratnam and Hurtig [9], who employed 3 types of screen arrangements: single, double, and triangular. They then investigated the effects of screen arrangement on free, imposed, and, in some cases, submerged hydraulic jumps.

Their experimental results covered a range of Froude number between 4 and 13 and showed that screens or porous baffles with porosity of 40% were able to dissipate more energy compared to that of free hydraulic jumps [9]. Çakır [10] and Bozkus et al. [11,12], in their experimental study on energy dissipation, introduced several factors like porosity, thickness, and location of screens, along with Froude numbers from 5 to 18. Çakır [10] and Bozkus et al. [11,12] suggested 40% as the average optimum screen porosity.

According to the studies in the literature [9-12], the screen performance (the ratio of the dimensionless parameter of energy dissipation) decreases with increase in relative location of the screens. A double screen arrangement would have more energy loss compared to a single arrangement [10-12]. Balkıs [13] and Bozkus et al. [14,15] investigated the effect of inclination of the screen on energy dissipation. The arrangement of screens in their experiments resembled those in [10-12] and they varied the Froude number from 5 to 24. The experiments of Balkıs [13] and Bozkus et al. [14,15] showed that the inclination parameter has an insignificant effect on the energy dissipation through a screen.

Aslankara [16] and Bozkus and Aslankara [17] investigated the effects of tailwater, with Froude numbers ranging from 5 to 22.5, on energy dissipation. Their results showed that tailwater has no significant additional contribution on energy dissipation. However, multiple screen arrangement has more energy dissipation compared to a single screen arrangement [16,17].

Most of the existing studies in the literature generally concentrated on imposed hydraulic jumps with high Froude numbers. However, the submerged hydraulic jump, which is typical in small hydraulic structures, has not earned equal attention. This is because oscillations take place in submerged hydraulic jumps. Screens creating submerged hydraulic jump with low Froude number as energy dissipaters have not been thoroughly studied. This study intends to fill this gap.

This paper, in addition to investigating free and imposed hydraulic jumps, mainly focuses on energy dissipation by screens in a submerged hydraulic jump with relatively low Froude numbers ranging from 2.5 to 8.5.

2. Screens as energy dissipaters

Depending upon the flow conditions and screen type and placements, it was possible to observe different behaviors.

2.1. Various types of behavior through collision of the supercritical flow with screens

When a screen was located in the direction of supercritical flow, 3 types of behavior were observed. 1) When the screen was located in the subcritical region and the distance between screen and the gate was more than the length of a full hydraulic jump, the hydraulic jump occurred before the screen (Figure 1a), agreeing with Çakır [10]. 2) When the screen was placed in the supercritical region, the collision of the supercritical flow with the screen imposed hydraulic jump. However, in this case, the distance between the jump point and screen

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was so small that a full jump could not take place. Still, due to the turbulences, the energy dissipation in the second case was more than in the first (Figure 1b), agreeing with Balkıs [13]. 3) When the Froude number of supercritical flow was relatively low or when the tailwater was relatively high, the collision of supercritical flow with the screens led to submerged hydraulic jump. The screen and the flow that left the gate were both located at the subcritical region (Figure 1c), agreeing with Aslankara [16] and Sadeghfam [18].

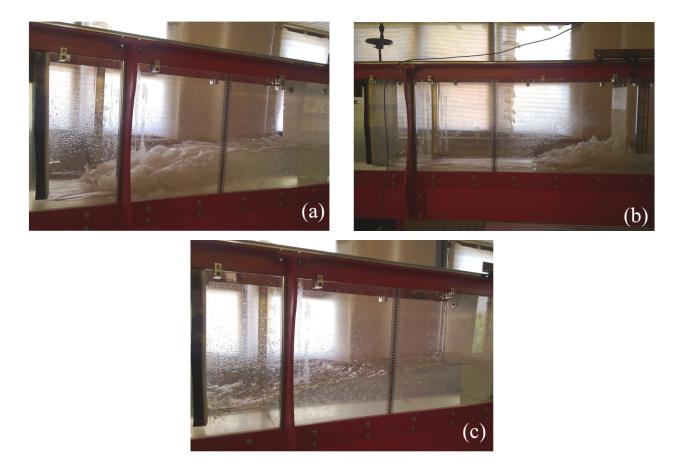


Figure 1. Different behaviors through collision of supercritical flow with screens: (a) first type, (b) second type, (c) third type.

2.2. Calculation of energy dissipation for the 3 cases

Figures 2a, 2b, and 2c are respectively the schematic illustrations for the first, second, and third behavior of supercritical flow it when encounters a screen. The energy loss between sections A and B in the first and second behavior can be calculated by using the energy principle [19]:

$$\Delta E_{AB} = E_A - E_B = \left(y_A + \frac{V_A^2}{2g}\right) - \left(y_B + \frac{V_B^2}{2g}\right),\tag{1}$$

where V_A and y_A are respectively velocity and flow depth at section A, V_B and y_B are respectively velocity and depth at section B, and g is acceleration due to gravity.

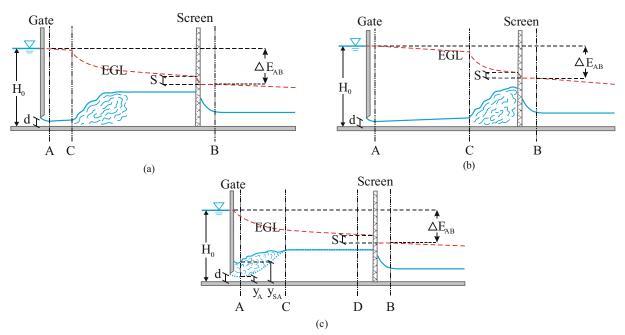


Figure 2. Different behaviors through collision of supercritical with screens: (a) first behavior, (b) second behavior, (c) third behavior [18].

The velocity at sections A and B is obtained by computing the average velocity during experiments. The flow depth at section A is obtained by Eq. (2) while the depth at section B is measured:

$$y_A = d \times C_C,\tag{2}$$

where d is the gate opening and C_C is the contraction coefficient.

According to Figure 2c, the energy loss between sections A and B, which includes the energy loss through submerged hydraulic jump and the screen, is calculated by the energy principle, i.e. by Eq. (1). The distinctive point in the third behavior is the depth and velocity at section A. Measuring the flow depth at section A or submerged gate depth (y_{SA}) is not possible because of the turbulences through the hydraulic jump. Rajaratnam and Hurtig [9] calculated this value by Eq. (3) [19], as follows:

$$y_{SA} = H_o - \frac{V_A^2}{2g},$$
(3)

where H_0 is the water depth behind the gate. For determining the submerged gate depth in the submerged hydraulic jump, Chow [19] presented Eq. (4) (Figure 2c):

$$y_{SA} = y_D \sqrt{1 + 2Fr_D^2 \left(1 - \frac{y_D}{y_A}\right)},\tag{4}$$

where y_A and y_D are respectively the water depth at A and D sections (Figure 2c), and Fr_D is the Froude number of the flow at section D and is calculated by $Fr_D = \frac{V_D}{\sqrt{gy_D}}$.

Finally, the energy loss at sections A to B is computed by using Eq. (5) [19]:

$$\Delta E_{AB} = \left(y_{SA} + \frac{V_A^2}{2g}\right) - \left(y_B + \frac{V_B^2}{2g}\right).$$
(5)

The contraction coefficient (C_c) in Eq. (2) for the free and submerged flows is calculated by the analytical results of Belaud et al. [20], through the ratio of gate opening to the depth of the approaching flow behind the gate (Figure 3).

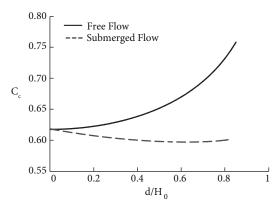


Figure 3. The contraction coefficient presented for submerged and free flows by Belaud et al. [20].

Admittedly, using the results of the work of Belaud, the value of C_C in this study varies in the interval of 0.614 to 0.628. It is worth noting that a constant value of 0.625 for the contraction coefficient (C_C) were considered in [10–17].

3. Dimensional analysis

Flow between the gate and the screen was theoretically analyzed and the following parameters were identified to perform the dimensional analysis [10,13,16,18]:

$$S = f_1(Q, d, w, y_A, y_B, y_C, x, X, p, G, t, g, \rho, \mu),$$
(6)

where S is energy dissipation due to the screen, whose dimension is [L]. Other parameters of above equations $(Q, d, w, y_A, y_B, y_C, x, X, p, k, t, g, \rho, \text{ and } \mu)$ and their dimension are, respectively, discharge $[L^3T^{-1}]$, gate opening [L], width of channel [L], water depth at section A [L], water depth at section B [L], water depth at section C [L], distance from upstream end of the imposed hydraulic jump to the screen [L], distance between the screen and the gate [L], porosity of screen, gaps between screens [L], thickness of the screen [L], gravitational acceleration $[LT^{-2}]$, density of water $[ML^{-3}]$, and dynamic viscosity of water $[ML^{-1}T^{-1}]$.

Energy at section A symbolized by E_A , length of free hydraulic jump (L), and Froude number at section B (Fr_B) can be described as the following functions:

$$E_A = f_2\left(Q, d, w, y_A, g\right),\tag{7}$$

$$L = f_3\left(Q, w, y_A, g\right),\tag{8}$$

$$Fr_B = f_4(Q, w, y_B, g).$$
(9)

Thus, substituting w, y_A , and y_B into Eq. (6) for E_A , L, and Fr_B respectively leads to:

$$S = f_5(Q, d, E_A, L, Fr_B, y_C, x, X, p, G, t, g, \rho, \mu).$$
(10)

By selecting y_A , g, and ρ as repeating variables, the following nondimensional equation is obtained:

$$\frac{S}{y_A} = f_6\left(\frac{E_A}{y_A}, Fr_A, \frac{L}{y_A}, Fr_C, \frac{x}{y_A}, \frac{X}{y_A}, p, \frac{G}{y_A}, \frac{t}{y_A}, \frac{y_A}{d}, Re\right),\tag{11}$$

in which Re is the Reynolds number. By reorienting Eq. (10), it can be represented in the following dimensionless format:

$$\frac{S}{E_A} = f_7 \left(Fr_A, p, \frac{G}{t}, \left| Fr_B, \frac{E_A}{d}, \frac{x}{d}, \frac{x}{L}, \frac{X}{d}, C_C, Re, \frac{t}{d} \right| \right).$$
(12)

It is worth noting that the parameters Fr_B , $\frac{E_A}{d}$, $\frac{x}{d}$, $\frac{X}{d}$ and $\frac{x}{L}$, which can be considered as case studies in future works, are out of the scope of the current study. As far as the contraction coefficient C_C , which is a function of d/H_0 and constant for each case of the experiment, is concerned, its value was obtained in a research work done by Belaud et al. [20]. Since in our investigation the type of flow is free surface flow, in which gravitational effects are more dominant, the importance of the Reynolds number is secondary and it resultantly is neglected. The thickness parameter of screens was introduced as an insignificant one in the literature [10,13,14], so $\frac{t}{d}$ is assumed to be a negligible parameter.

By taking all the mentioned justifications into account and dropping the discussed parameters of the previous paragraph, the final dimensionless equation is obtained as represented in Eq. (13). As can be seen, the Froude number of supercritical flow (Fr_A) , porosity of screen (p), and arrangement of double screen $(\frac{G}{t})$ play an important role in the current study.

$$\frac{S}{E_A} = f_7\left(Fr_A, p, \frac{G}{t}\right) \tag{13}$$

4. Experimental set-up and experiments

The experiments in the current study were performed in a horizontal open channel (flume) of 0.3 m wide, 0.5 m deep, and 5 m long at Maragheh University in Iran (Figure 1). Figure 4 presents the detailed schematic representation of the experimental set-up. The inlet discharge was supplied by 2 pumps that were fed by channel tank, and the inlet flow was controlled by 2 valves that were installed in the outlet of the pumps (Figure 4). The inlet discharge was measured by 2 rotameters, which were installed in the outlet of the pumps. The rotameters were calibrated by the accurate measurement of a velocity meter in subcritical flow at numerous depths and discharges. Additionally, in this step, turbulence and uniformity of the inlet velocity profile were controlled. The gate was installed at a distance of 1 m from the inlet, and the opening was fixed at 2.6 cm. The Froude number of supercritical flow was varied from about 2.5 to 8.5. With the assumption that there was no energy loss between the gate outlet and vena contracta, all the initial energy calculations were done with respect to the depth at the vena contracta.

The experiments were performed in 2 series. The first was conducted by measuring the depth by using a point gauge, and the second was conducted by digital measurement equipped with ultrasonic sensors and an accuracy of 140 data per second. In the first series of experiments, the flow depths were measured by point gauge through calculating the average number of 3 depths in the middle and 7.5 cm from the channel walls. The equations of Rajaratnam and Subramanya [21] and Chow [19] were used to calculate the depth of the submerged gate. In the second series of experiments, all depths and especially the depth of the submerged gate were measured by digital measurement in 45 s. The use of digital measurement was very efficient in measuring the water surface profile.

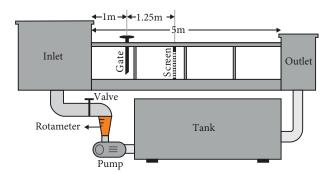


Figure 4. Schematic picture of experimental set-up.

The screens used in the experiments were made of polyethylene and each screen had a thickness of 1 cm. Porosities of 40% and 50% were created by holes of 1 cm in diameter through uniform square meshes. The screens had single and double arrangements, which were created by gaps of 1, 3, and 5 cm. Like in the experiments of Rajaratnam and Hurtig [9], the screens were fixed at a distance of 1.25 m from the gate.

5. Analysis of experimental results

In this section, quite a few experiments are devised to analyze the behavior of a supercritical flow when it encounters screens, which cause submerged hydraulic jumps generated upstream of the screens.

In the analysis of the experimental results, 2 parameters, namely percent energy loss (%) and Fr number, were evaluated for the screen performance. Figures 5–8 respectively compare performances for a single screen with porosity of 40% and 50% and double screens with 1 cm gap and porosity of 40% and 50%. In these figures, parts a and b illustrate, respectively, performance of screens with regard to upstream and downstream. The digital measurements were compared with the results of equations of Chow [19] and Rajaratnam and Subramanya [21]. According to Figures 5–8, it can be inferred that the equation of Rajaratnam performs satisfactorily in simulating the submerged hydraulic jump. It is worth noting that by increasing the Froude number the performance of screens in dissipating the energy loss due to the classical free hydraulic jump. As seen, the energy loss through screens for all conditions is more than the energy loss through free hydraulic jump.

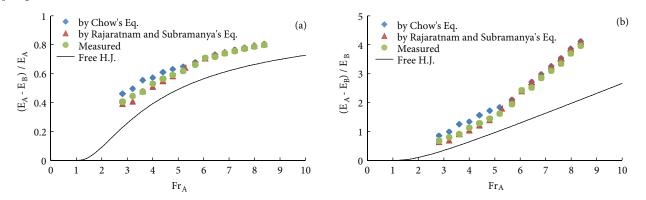


Figure 5. The energy loss of single screens with the porosity of 40%: (a) performance of screens with regard to upstream, (b) performance of screens with regard to downstream.

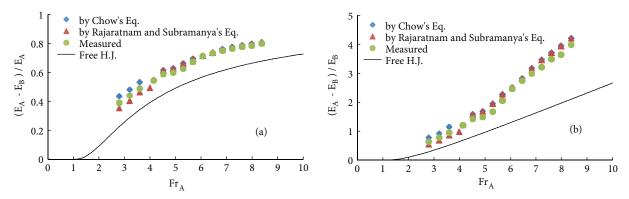


Figure 6. The energy loss of single screens with porosity of 50%: (a) performance of screens with regard to upstream, (b) performance of screens with regard to downstream.

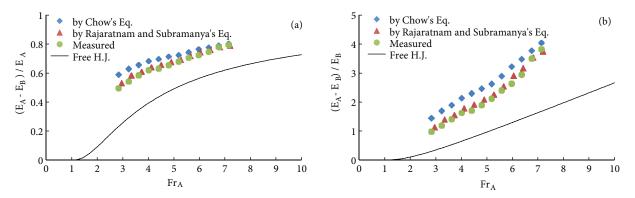


Figure 7. The energy loss of double screens with porosity of 40%: (a) performance of screens with regard to upstream, (b) performance of screens with regard to downstream.

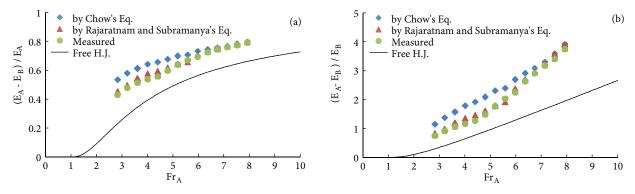


Figure 8. The energy loss of double screen with porosity of 50%: (a) performance of screens with regard to upstream, (b) performance of screens with regard to downstream.

Figures 9–12 present effects of single screen, double screen, porosity, and different screen gaps on energy loss as a function of Fr number. In Figures 9–11, parts a and b represent, respectively, performance of screens with regard to upstream and downstream. SS and DS respectively stand for single screen and double screen. P stands for porosity and G stands for gap.

Figure 9 compares the results of a single screen with porosity of 40% and 50%. Based on this figure, the results of the screen with porosity of 40% and 50% are correlated. Rajaratnam and Hurtig [9] claimed that

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40% porosity is the optimum rate. However, according to Figure 9, this is not exactly the case. This is because of the different flow conditions in 2 cases. In the Froude number range in this study, the supercritical flow in collision with single screens with porosity of 50% shows the imposed hydraulic jump (second behavior) and the second behavior creates more energy loss due to the extra turbulences compared to the first (free hydraulic jump) and third (submerged hydraulic jump) behaviors.

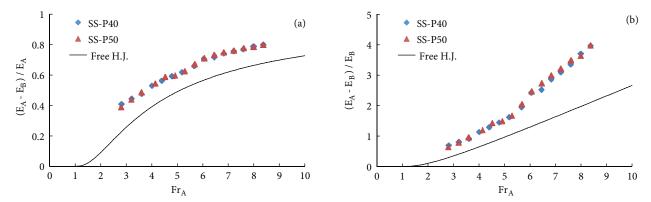


Figure 9. Comparison of the screen results of the porosity of 40% and 50%: (a) the performance of the screens with regard to upstream, (b) performance of the screens with regard to downstream.

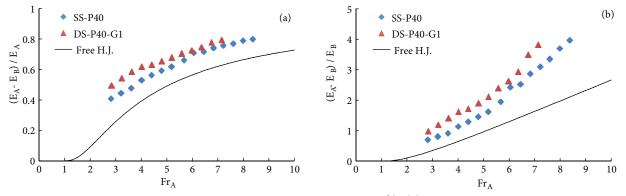


Figure 10. Comparison of single and double screens with porosity of 40%: (a) performance of the screens with regard to upstream, (b) performance of the screens with regard to downstream.

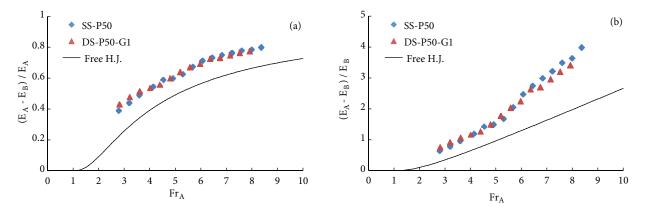


Figure 11. Comparison of the single and double screens with porosity of 50%: (a) performance of the screens with regard to upstream, (b) performance of the screens with regard to downstream.

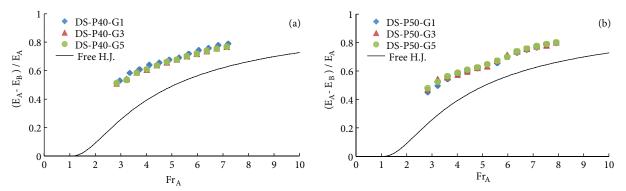


Figure 12. Comparison of the screens with double arrangement: (a) performance of the screens with regard to upstream with porosity of 40%, (b) performance of the screens with regard to upstream with porosity of 50%.

Figures 10 and 11 respectively compare the results of single and double screens with porosity of 40% and 50% for both upstream and downstream flow conditions. The double arrangement of screens with 40% porosity has more energy loss compared to single arrangements (Figure 10). The energy loss of the single arrangement case is almost the same as that of the double arrangement for 50% porosity (Figure 11).

Figures 12a and 12b compare the results of the double screen arrangement with gaps of 1, 3, and 5 cm with porosity of 40% and 50%, respectively. According to Figure 12, the gap between screens in the study range (Fr number range: 2.5–8.5) has an insignificant effect on energy dissipation. Figure 13 also illustrates some comparisons between existing studies and the current one. Based on this figure, it is observed that the results are approximately within the same range.

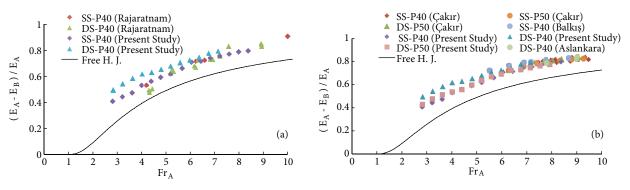


Figure 13. The comparison of the current and former studies.

Based on the experimental results in this study, nonlinear regression equations (quadratic polynomial) were developed as a function of Fr number. The general forms of these equations are presented in Eqs. (14) and (15):

$$(E_A - E_B)/E_A = P_1(Fr_A)^2 + P_2(Fr_A) + P_3,$$
(14)

$$(E_A - E_B)/E_B = P_1(Fr_A)^2 + P_2(Fr_A) + P_3,$$
(15)

where P_1 , P_2 , and P_3 are constants (coefficients). These equations indicate the importance of the Froude number in energy dissipation.

Tables 1 and 2 present the obtained optimal values of the parameters of the equations for each case. The optimal values of the coefficients in Tables 1 and 2 were obtained by regression method using MATLAB. The

last column in these tables presents the relative energy dissipation computed by Eq. (16). The high value of this parameter shows the high value of energy dissipation with regard to free hydraulic jump:

$$\Delta E_{(Coulmn7)} = \frac{\Delta E_{Screen} - \Delta E_{FreeJump}}{\Delta E_{FreeJump}},\tag{16}$$

where ΔE_{Screen} and $\Delta E_{FreeJump}$ are, respectively, energy dissipation through screens and free hydraulic jump. Based on the values in the last column (column 7) in the tables, energy dissipation through screens is higher than free hydraulic jump. Furthermore, for equal porosity, the double arrangement with 40% porosity dissipates more energy compared to the single arrangement.

						ΔE compared
	P_1	P_2	P_3	RMSE	R-square	to free
						jump (%)
SS-P40	-0.007573	0.1569	0.01853	0.0077	0.9970	26.81
SS-P50	-0.009196	0.1763	-0.03115	0.009112	0.996	27.41
DS-P40-G1	-0.005495	0.114	0.2537	0.007896	0.9923	40.60
DS-P40-G3	-0.006282	0.1212	0.2205	0.004553	0.9976	32.15
DS-P40-G5	-0.006647	0.1254	0.2099	0.005041	0.9971	36.68
DS-P50-G1	-0.004835	0.1179	0.1679	0.009284	0.9939	32.06
DS-P50-G3	-0.002943	0.09407	0.2446	0.01358	0.9854	33.80
DS-P50-G5	-0.003299	0.0979	0.2393	0.009478	0.9928	34.59

Table 1. The constants of fitted equations for screen performance with regard to upstream.

Table 2. The constants of fitted equations for screen performance with regard to downstream.

	P_1	P_2	P_3	RMSE	R-square	ΔE compared to free jump (%)
SS-P40	0.02998	0.2847	-0.4501	0.0912	0.9942	84.98
SS-P50	0.03649	0.2159	-0.3072	0.1062	0.9922	83.34
DS-P40-G1	0.05991	-0.0007363	0.6828	0.06684	0.995	133.95
DS-P40-G3	0.03667	0.1565	0.3217	0.03254	0.9984	110.31
DS-P40-G5	0.03225	0.2039	0.2076	0.03135	0.9986	110.26
DS-P50-G1	0.07043	-0.1573	0.7691	0.0724	0.9956	96.81
DS-P50-G3	0.07414	-0.1981	0.9333	0.1105	0.9897	102.35
DS-P50-G5	0.08058	-0.25	1.055	0.09071	0.9934	106.79

The obtained equations for screen performance with regard to upstream and downstream are plotted, respectively, in Figures 14a and 14b. The experimental results are not shown for clarity. Figure 14 and the very low RMSE and very high R^2 values in the tables indicate the excellent fit of the equations to the measured data. Hence, these equations can be confidently employed for interpolation and/or extrapolations purposes.

6. Conclusions

This paper presented an experimental study on the energy dissipation through screens. The Froude number of supercritical flow was varied from about 2.5 to 8.5 and screens with porosities of 40% and 50% with single and double arrangements and gaps of 1, 3, and 5 cm were investigated. The main focus of the current paper

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was on the submerged hydraulic jump. The amount of energy loss through submerged jumps was gauged by digital measurements and then the results were compared with 2 conventional equations (namely Chow's and Rajaratnam's equations).

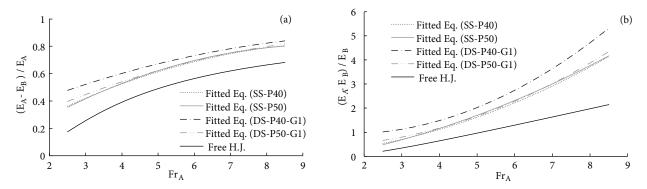


Figure 14. Fitted equations for different arrangement of screens: (a) performance of the screens with regard to upstream, (b) performance of the screens with regard to downstream.

According to the experimental results, the screens have more energy loss compared to free hydraulic jumps, and the amount of energy dissipation increases with the increasing of the Froude number. The double screen arrangements have more energy dissipations compared to the single arrangement within the study range. The gaps of the double arrangement have an insignificant effect on energy dissipation. Additionally, in the second behavior (imposed hydraulic jump), there is more energy loss compared to the 2 other cases (free and submerged hydraulic jump).

The quadratic polynomial equations were fitted to the results of screen performance with acceptable values of \mathbb{R}^2 and RMSE. These equations, which can be used for interpolation and extrapolation purposes, indicate that energy dissipation through screens is mainly a function of Froude number.

Based on computing energy dissipation through screens compared to free hydraulic jump, the double screen with porosity of 40% has the best performance.

The presented method is mostly useful for weak and medium jumps, not strong jumps that happen in dam weirs. This method generally is suggested for the regime transformation in the conveyance channels at the drops, entrance of basins, and settling basin. Because in dam weirs the possibility of blockage of holes and consequently overtopping occurrence is inevitable, its usage in the USBR type 1, type 2, and type 3 and flip bucket is recommended. As an application in a stilling basin, where a screen network is installed right before the entrance to the basin, this device (screen network) can be beneficial in 2 ways: omission of floating objects and transforming of the flow regime. In addition, in flow conveyance channels through setting the screen network after the drops, there will be no need for stilling the basin. The restriction in installation of screen networks in dams is the main disadvantage of this devise.

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